

Applications of Passivity Theory to the Active Control of Acoustic Musical Instruments

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Introduction

Passivity For Linear Systems

PID Control

Other Passive Linear Controllers

Nonlinear PID Control



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- ▶ We use a digital feedback controller.
- ▶ The resulting instrument is like a haptic musical instrument whose interface is the whole acoustical medium.
- ▶ We apply the technology to a vibrating string, but the controllers are applicable to any passive musical instrument.



System Block Diagram

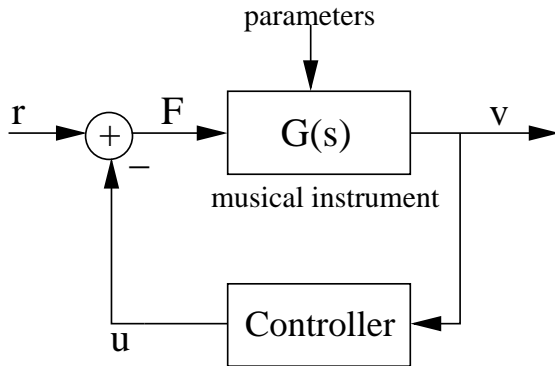


Figure: System block diagram for active feedback control

- ▶ We would like the controller to be robust to changes in $G(s)$.



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- ▶ Note: The bilinear transform preserves s-domain and z-domain sense positive realness.



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- ▶ This property is known as unconditional stability.



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- ▶ and the envelope of the impulse response decays exponentially with time constant $\tau = 2m/R$.



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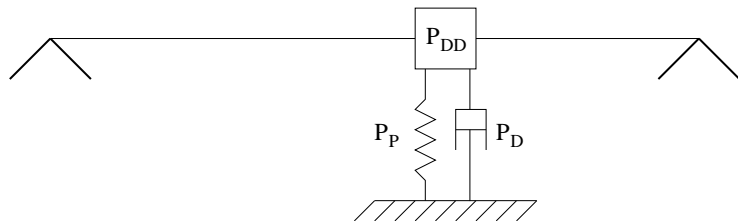
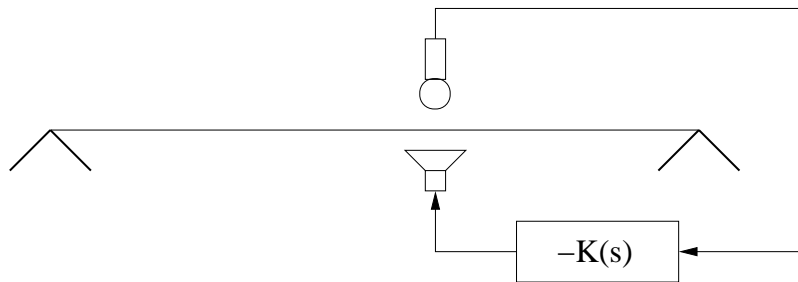
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PID Control Mechanical Equivalent



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 5. Filter alternating between $\pm\pi/2$ radians



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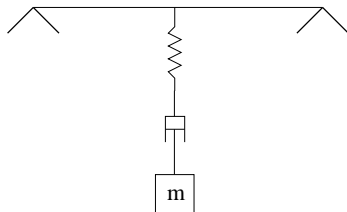
- ▶
$$K_{bp}(s) = \frac{\frac{\omega_c s}{Q}}{s^2 + \frac{\omega_c s}{Q} + \omega_c^2}$$



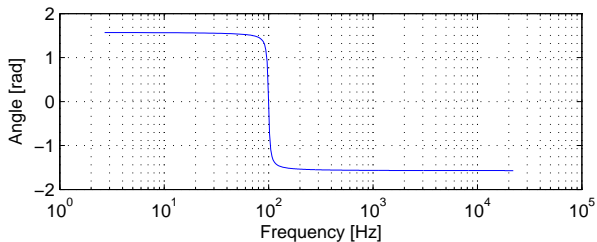
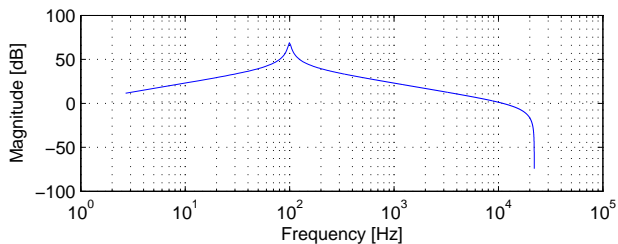
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Bandpass Filter



Notch Filter Control

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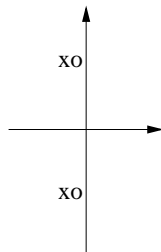
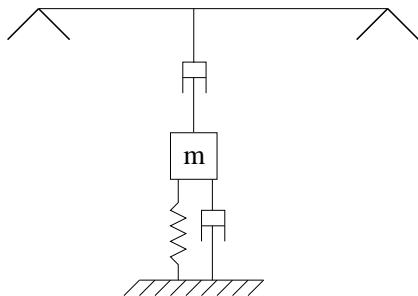
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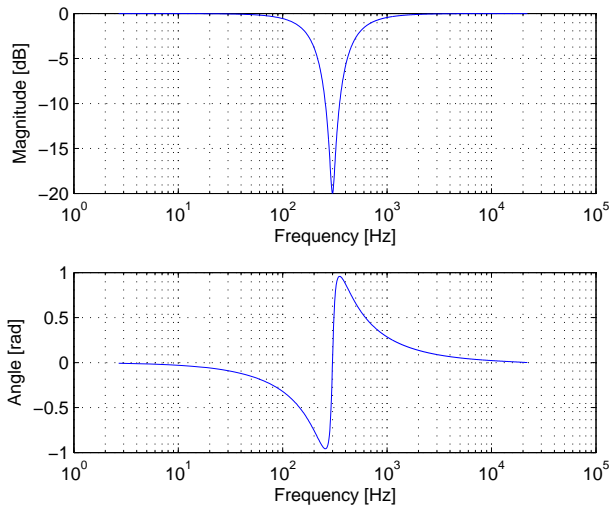


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$$K_{notch}(s) = \frac{s^2 + \frac{\omega_c s}{\alpha Q} + \omega_c^2}{s^2 + \frac{\omega_c s}{Q} + \omega_c^2}$$

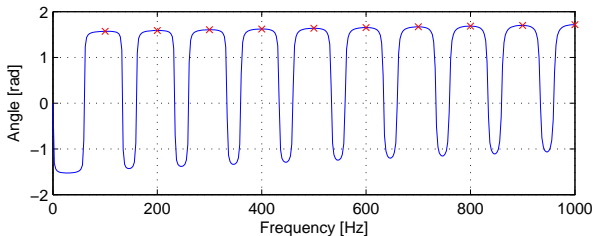
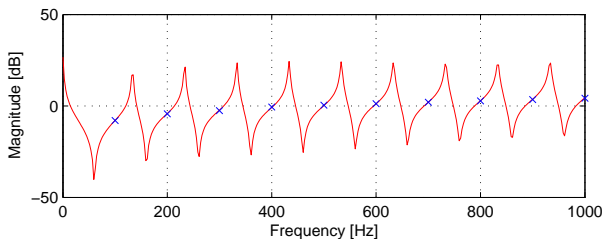


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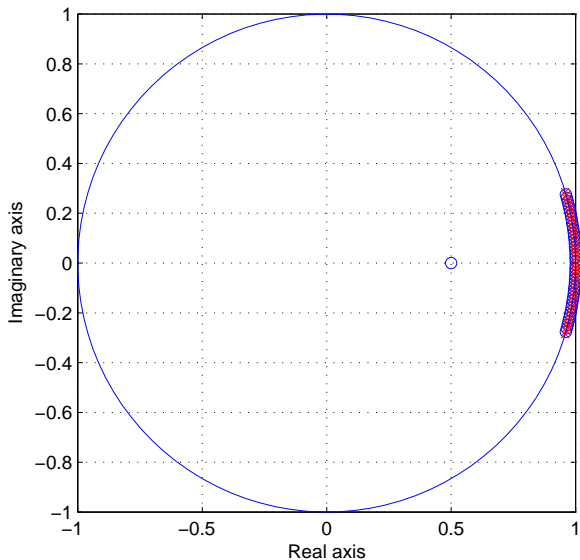


Alternating Filter

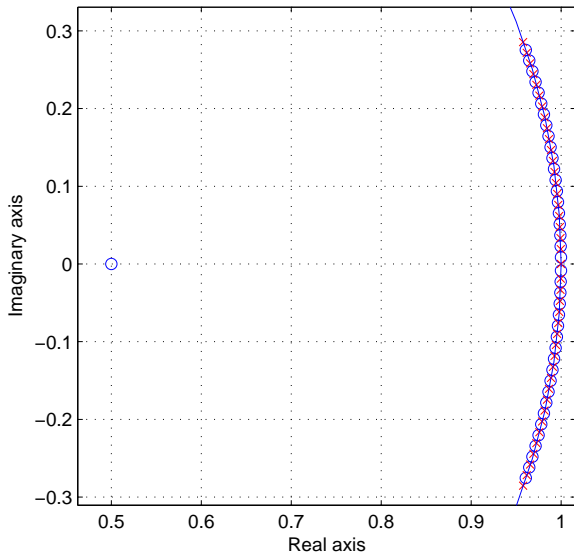
- ▶ The frequency response shown below is such that partials at $n100\text{Hz}$ (shown by x's) will be pushed flat.



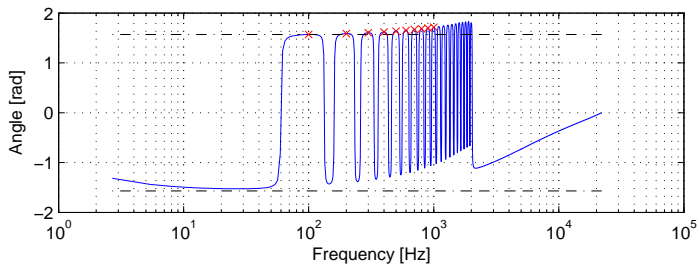
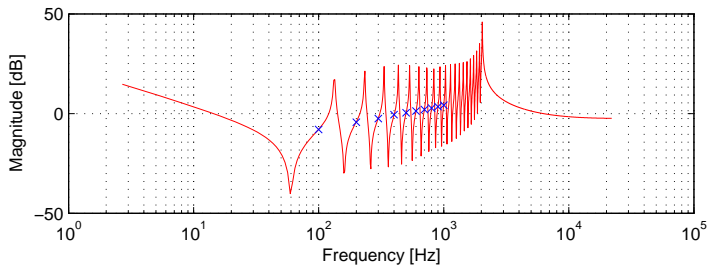
Alternating Filter Implementation



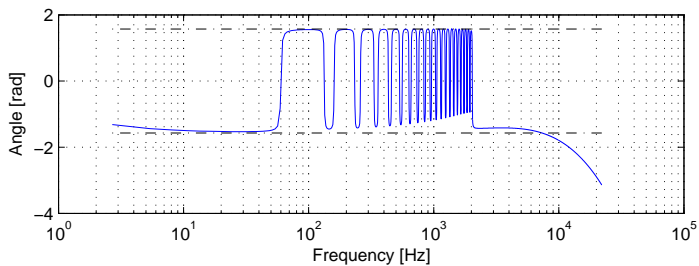
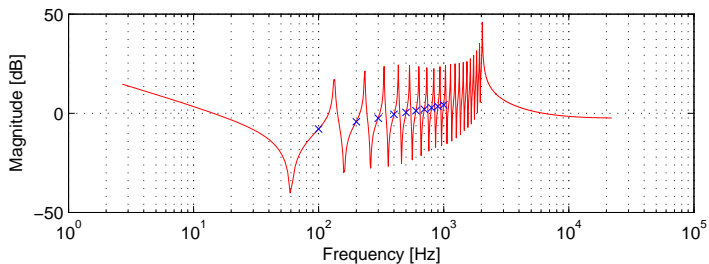
Alternating Filter Implementation (Zoomed)



Wideband Idealized Frequency Response



Wideband Frequency Response Including Delay



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$$m\ddot{x} + R(\dot{x}, x) + K(x) = 0 \quad (6)$$



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- ▶ There are many methods for analyzing the behavior of second-order nonlinear systems.



Linear Dashpot

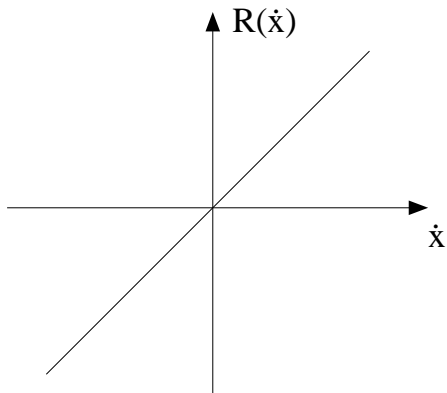


Figure: Linear Dashpot



Linear Dashpot

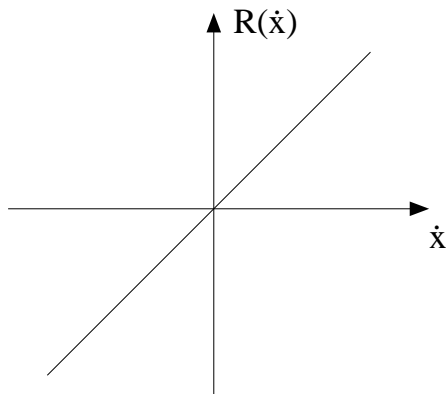


Figure: Linear Dashpot

- ▶ $R(\dot{x}, x) = R\dot{x}$ for some constant R



Saturating Dashpot

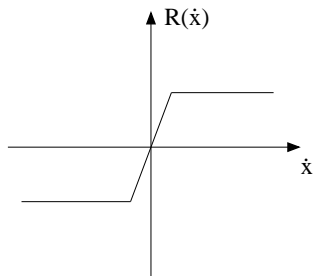


Figure: Saturating Dashpot



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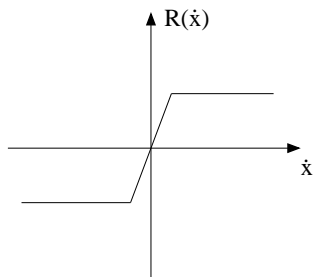


Figure: Saturating Dashpot

- ▶ Damping is *passive* if $\dot{x}R(\dot{x}, x) \geq 0$ for all \dot{x} and x .
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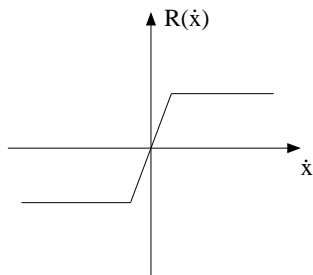


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 - ▶ $R(\dot{x}, x) > 0$ for $\dot{x} > 0$
 - ▶ $R(\dot{x}, x) < 0$ for $\dot{x} < 0$
- ▶ Damping is *strictly passive* if $\dot{x}R(\dot{x}, x) > 0$ for all x and for all $\dot{x} \neq 0$ (i.e. there is no deadband).



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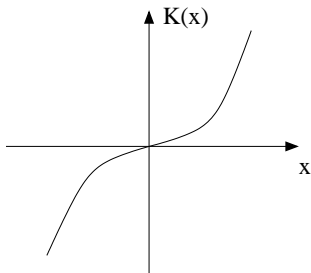


Figure: Stiffening Spring



- ▶ A linear spring behaves according to $K(x) = kx$ for some constant k .

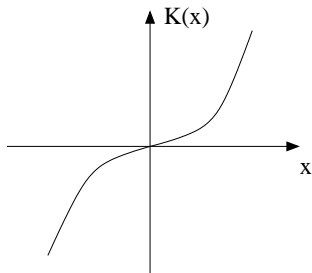


Figure: Stiffening Spring

- ▶ The spring is *strictly locally passive* if $xK(x) > 0 \forall x \neq 0$.



- ▶ The system $m\ddot{x} + R(\dot{x}, x) + K(x) = 0$ is stable if both the spring and dashpot are strictly locally passive.



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- ▶ You can prove this using the Lyapunov function

$$V = \frac{1}{m} \int_0^x K(\sigma) d\sigma + \frac{1}{2} \dot{x}^2 \quad (7)$$



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$$\dot{V} = -\frac{R(\dot{x}, x)\dot{x}}{m} \leq 0 \quad (8)$$



Nonlinear Dashpot for Bow at Rest

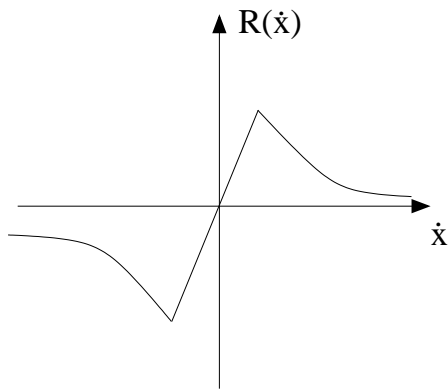


Figure: Bowing Nonlinearity



Nonlinear Dashpot For Moving Bow

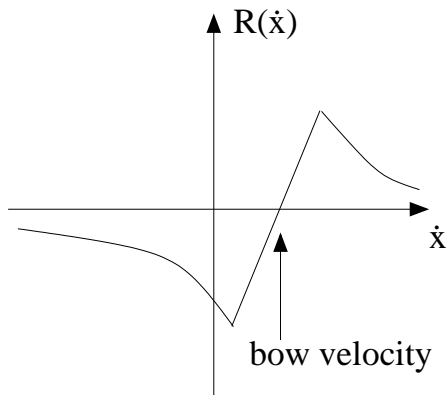


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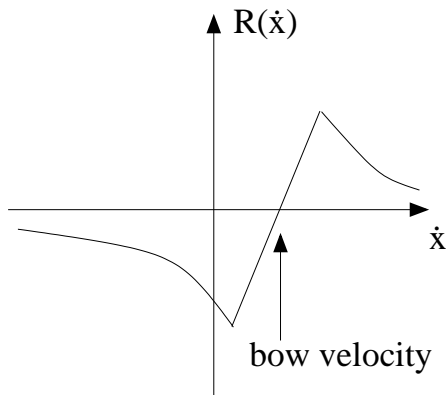


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- ▶ Now the dashpot is *NOT* passive.



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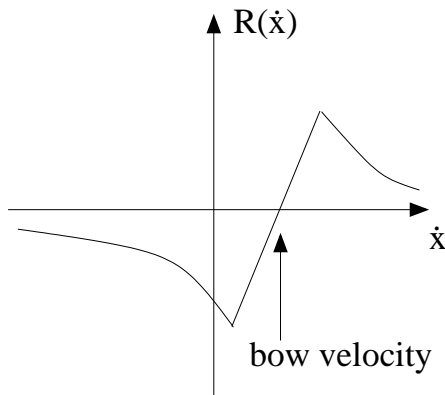


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- ▶ Now the dashpot is *NOT* passive.
- ▶ The negative damping region adds energy so that the bowed instrument can self-oscillate.



Thanks!

- ▶ Sound examples are on the website
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- ▶ Sound examples are on the website
<http://ccrma.stanford.edu/~eberdahl/Projects/PassiveControl>
- ▶ Questions?



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